

# Modified Discrete Sliding mode Model Following Control (DSMFC) for an Induction Motor Drivers \*

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## Abstract

A Discrete Sliding mode Model Following Control or DSMFC strategy for an induction motor drivers is presented. The DSMFC algorithm uses the combination of model following control and sliding mode control to improve the dynamics response for command tracking. A procedure is proposed for choosing the control function so that it guarantee the existence of a sliding mode. The DSMFC approach has been implemented on a microcontroller and applied to a position control of an induction motor. Experimental results illustrate that DSMFC gives a significant improvement on the tracking performances and robust to plant parameter variations and disturbances.

**Keywords:** Sliding Mode Control, Discrete Time System, Induction Motor Drivers.

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## Introduction

Recently, industry application of the high performance vector-controlled induction motor drivers have greatly increased, including machine tools, robotics and paper machines [P.C.Krause *et al.* 1995]. The ultimate object of vector control is to enable decoupling control of torque and flux similar to the control of a separately excited DC motor. However, the dynamic position vector control is sensitive to machine parameter variations. Moreover, the machine parameters and load characteristics are not exactly known, which way very during motor operation. Thus the dynamic of such system are complex and nonlinear, where the linear controller design may not assure satisfactory performances.

To satisfy these requirement, a variable structure control (VSC) or Sliding Mode Control (SMC) is invariant to system parameter variations and disturbances when the sliding mode occurs [U.Itkis, 1976 and V.I.Utkin *et al.* 1999], but it may result in a steady state error when there is load disturbance in it. In order to improve the problem, the Integral Variable Structure Model Following Control (IVSMFC) has been proposed [T.L.Chern *et al.* 1997 and T.L.Chern *et al.* 1998] for achieving a zero steady state error for step command input. The IVSMFC approach combines an integral controller with VSC and applied to the design of a Model Following Control (MFC). However, when command input is changing, e.g., ramp command input, the IVSMFC gives a steady state error. The Modified Integral Variable Structure Control or MIVSC approach, proposed in [S.Nungam and P.Daengkua. 1998], uses a double integral action to solve this problem. Although, the MIVSC method can give a better tracking performance than the IVSMFC method does at steady state, its performance during transient period needs to be improved.

In this paper, the Discrete Sliding mode Model Following Control or DSMFC approach is presented. This approach, which is the extension of FIVSC approach [P.Phakamach and S.Nungam. 1999], incorporates a feedforward path to improve the dynamics response for command tracking. The advantage of the DSMFC approach is that the error trajectory in the sliding motion can be prescribed by the design. Also, it can achieve a rather accurate servo tracking and is fairly robust to plant parameter variations and external load disturbances. An experimental prototype system consisting of a MCS-196MC microcontroller is constructed. As a results, the tracking performance can be remarkably improved. The DSMFC method is applied to an induction motor for a position control system.

## Design of DSMFC System

The structure of DSMFC system is shown in Figure 1. It combines the conventional VSC with a double-integral compensator, a feedforward path from the input command, a reference model and a comparator.

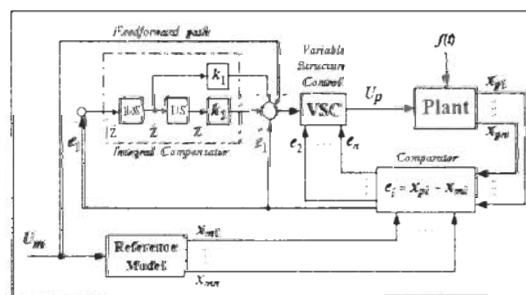


Figure 1 The structure of DSMFC system

Let the plant be described by the following equation :

$$\dot{x}_{pi} = x_{p(i+1)} \quad ; i=1, \dots, n-1, \quad \dot{x}_{pn} = -\sum_{i=1}^n a_{pi} x_{pi} + b_p U_p - f(t) \quad (1)$$

where  $a_{pi}$  and  $b_p$  are the plant parameters;  $f(t)$  are disturbances and  $U_p$  is the control input of the plant.

The reference model is represented by

$$\dot{x}_{mi} = x_{m(i+1)} \quad ; i=1, \dots, n-1, \quad \dot{x}_{mn} = -\sum_{i=1}^n a_{mi} x_{mi} + b_m U_m \quad (2)$$

where  $U_m$  is the input command of the system.

Defining  $e_i = x_{pi} - x_{mi}$  ;  $(i=1, \dots, n)$  and subtracting (2) from (1), the error differential equation is

$$\dot{e}_i = e_{i+1} \quad ; i=1, \dots, n-1, \quad \dot{e}_n = -\sum_{i=1}^n a_{pi} e_i - \sum_{i=1}^n (a_{mi} - a_{pi}) x_{mi} + b_m U_m - b_p U_p - f(t). \quad (3)$$

Using the SMFC approach [P.Phakamach and S.Nungam, 2000., P.Phakamach and S.Nungam, 2001] to the error dynamics in order to synthesise the control signal,  $U_p$  and assuming the asymptotic divergence of the error to zero, the DSMFC system in Fig. 1 can be described as

$$\ddot{z} = -e_1, \quad \dot{e}_i = e_{i+1} \quad ; i=1, \dots, n-1 \text{ and } \dot{e}_n = -\sum_{i=1}^n a_{pi} e_i - \sum_{i=1}^n (a_{mi} - a_{pi}) x_{mi} + b_m U_m - b_p U_p - f(t) \quad (4)$$

where  $U_p$  is the control function.

Consider the discretisation of the system given in (4). If the derivative is approximated by the forward difference as [C.I.Philips and H.T.Nagle, 2000]

$$\dot{e}_i(t) = \frac{e_i(t+T) - e_i(t)}{T}$$

where  $T$  is the sampling interval, then the discretised version of the system (4) can be represented as :

$$\dot{z}(k+1) = \dot{z}(k) - T e_1(k), \quad e_i(k+1) = e_i(k) + T e_{i+1}(k) \quad ; i=1, \dots, n-1 \quad (5a)$$

$$\dot{e}_n(k+1) = e_n(k) - T \sum_{i=1}^n a_{pi} e_i - T \sum_{i=1}^n (a_{mi} - a_{pi}) x_{mi}(k) + T b_m U_m(k) - T b_p U_p(k) - T f(k) \quad (5b)$$

The switching function,  $\sigma$  is given by

$$\sigma(k) = c_1 [e_1(k) - K_1 \dot{z}(k) - K_2 z(k) - r(k)] + \sum_{i=2}^n c_i e_i(k) \quad ; C_i > 0 = \text{constant}, C_n = 1. \quad (6)$$

In the discrete variable structure control system, the control input is computed at discrete instants and applied to the system during the sampling interval so that an ideal sliding motion cannot be obtained. The conditions ensuring the existence and reachability of a non-ideal sliding motion are :

$$\sigma(k) \Delta \sigma(k+1) < 0 \quad \text{and} \quad |\Delta \sigma(k+1)| < \frac{\xi}{2} \quad (7)$$

where  $\Delta\sigma(k+1) = \sigma(k+1)$ ;  $\sigma(k)$  and  $\xi$  is a small positive number. The control function can be chosen to guarantee that the inequalities (7) are satisfied, so a sliding mode motion control within the range of  $\xi$  will appear or  $\sigma(k) \leq \xi$ .

Design of such a system involves the choice of the control function  $U_p(k)$  to guarantee the existence of a sliding mode control, the determination of the switching function  $\sigma(k)$  and the integral control gain  $K_I$  such that the system has the desired properties and the elimination of chattering phenomena of the control input by using the smoothing function.

### Choice of control function

The control signal can be determined as follows. From (5) and (6), we have

$$\Delta\sigma(k+1) = -Tc_1(K_1\ddot{z}(k) + K_2\dot{z}(k)) + T\sum_{i=2}^n c_{i-1}e_i - \sum_{i=1}^n a_{pi}e_i(k) + \sum_{i=1}^n (a_{mi} - a_{pi})x_{mi}(k) - Tb_m U_m(k) + Tb_p U_p(k) - Tf(k) \quad (8)$$

Let  $a_{pi} = a_{pi}^0 + \Delta a_{pi}$ ;  $i=1, \dots, n$  and  $b_p = b_p^0 + \Delta b_p$ ;  $b_p^0 > 0$ ,  $\Delta b_p > -b_p^0$ ,

where  $a_{pi}^0$  and  $b_p^0$  are nominal values;  $\Delta a_{pi}$  and  $\Delta b_p$  are the associated variations.

Let the control function  $U_p(k)$  be decomposed into

$$U_p(k) = U_{eq}(k) + U_s(k) \quad (9)$$

where the so called equivalent control  $U_{eq}(k)$  is defined as the solution of (9) under the condition where there is no disturbances and no parameter variations, that is  $\Delta\sigma(k+1) = 0$ ,  $f(k) = 0$ ,  $a_{pi} = a_{pi}^0$ ,  $b_p = b_p^0$  and  $U_p(k) = U_{eq}(k)$ . This condition results in

$$U_{eq}(k) = \left\{ -c_1(K_1\ddot{z}(k) + K_2\dot{z}(k)) - \sum_{i=2}^{n-1} c_{i-1}e_i(k) + \sum_{i=1}^{n-1} a_{pi}^0 e_i(k) - \sum_{i=1}^n (a_{mi} - a_{pi}^0)x_{mi}(k) + b_m U_m(k) \right\} / b_p^0. \quad (10)$$

In the sliding motion,  $\sigma(k) = 0$ , one can obtain

$$e_n(k) = \left[ -c_1[e_1(k) - K_1\dot{z}(k) - K_2z(k) - r(k)] - \sum_{i=2}^n c_i e_i(k) \right]. \quad (11)$$

Substitution of (11) into (10) yields

$$U_{eq}(k) = \left\{ -c_1(K_1\ddot{z}(k) + K_2\dot{z}(k)) - T\sum_{i=2}^{n-1} c_{i-1}e_i(k) + T\sum_{i=1}^{n-1} a_{pi}^0 e_i(k) - T\sum_{i=1}^n (a_{mi} - a_{pi}^0)x_{mi}(k) + b_m U_m(k) \right. \\ \left. + (c_{n-1} - a_{pn}^0)[c_1(e_1(k) - K_1\dot{z}(k) + K_2z(k) - U_m(k)) + \sum_{i=2}^{n-1} c_i e_i(k)] \right\} / b_p^0. \quad (12)$$

The function  $U_s(k)$ , is employed to eliminate the influence due to  $\Delta a_{pi}$ ,  $\Delta b_p$  and  $f(k)$  so as to guarantee the existence of a sliding mode control, this function is constructed as

$$U_s(k) = \varphi_1(e_1(k) - K_1\dot{z}(k) - K_2z(k) - U_m(k)) + T \sum_{i=2}^n \varphi_i e_i(k) + \varphi_{n+1}. \quad (13)$$

Where :

$$\varphi_i = \begin{cases} \alpha_i & \text{if } [e_1(k) - K_1\dot{z}(k) - K_2z(k) - U_m(k)]\sigma(k) > 0 \\ \beta_i & \text{if } [e_1(k) - K_1\dot{z}(k) - K_2z(k) - U_m(k)]\sigma(k) < 0 \end{cases}, \quad \varphi = \begin{cases} \alpha & \text{if } e_i\sigma(k) > 0 \\ \beta & \text{if } e_i\sigma(k) < 0 \end{cases}; i = 2, \dots, n \text{ and}$$

$$\varphi_{n+1} = \begin{cases} \alpha_{n+1} & \text{if } \sigma(k) > 0 \\ \beta_{n+1} & \text{if } \sigma(k) < 0 \end{cases}.$$

Substitute (7) and (9) into (6), to obtain

$$\begin{aligned} \Delta\sigma(k+1)\sigma(k) = & T \{ [-\Delta a_{p1} + a_{p1}^0 \Delta b_p / b_p^0 + c_1(c_{n-1} - a_{pn}^0)(1 + \Delta b_p / b_p^0) + b_p \varphi_1](e_1(k) - K_1\dot{z}(k) - K_2z(k) - U_m(k))\sigma(k) \\ & + \sum_{i=2}^{n-1} \{ [-\Delta a_{pi} + a_{pi}^0 \Delta b_p / b_p^0 + c_{i-1} \Delta b_p / b_p^0 + c_i(c_{n-1} - a_{pn}^0)(1 + \Delta b_p / b_p^0) + b_p \varphi_i] e_i(k) \sigma(k) \} \\ & + [-\Delta a_{pn} + (c_{n-1} - a_{pn}^0) + b_p \varphi_n] e_n \sigma(k) + [N(k) + b_p \varphi_{n+1}] \sigma(k) \end{aligned} \quad (14)$$

where

$$N = -(K_1\dot{z}(k) + K_2z(k))(\Delta a_{p1} - a_{p1}^0 \Delta b_p / b_p^0) + \Delta b_p / b_p^0 [c_1(K_1\dot{z}(k) + K_2z(k))] + [-\sum_{i=1}^n (a_{mi} - a_{pi}^0) x_{mi}(k) + b_m U_m] \Delta b_p / b_p^0 - f(k).$$

In order for (7) to be satisfied, the following conditions must be met,

$$\begin{aligned} \varphi_i = & \begin{cases} \alpha_i & \langle \varphi_i = \text{Inf}[\Delta a_{pi} - a_{pi}^0 \Delta b_p / b_p^0 + c_{i-1} \Delta b_p / b_p^0 - c_i(c_{n-1} - a_{pn}^0)(1 + \Delta b_p / b_p^0)] / b_p \\ \beta_i & \langle \varphi_i = \text{Sup}[\Delta a_{pi} - a_{pi}^0 \Delta b_p / b_p^0 + c_{i-1} \Delta b_p / b_p^0 - c_i(c_{n-1} - a_{pn}^0)(1 + \Delta b_p / b_p^0)] / b_p \end{cases} \text{ where } i=1, \dots, n-1, c_0 = 0, \\ \varphi_n = & \begin{cases} \alpha_n & \langle \varphi_n = \text{Inf}[\Delta a_{pn} + a_{pn}^0 - c_{n-1}] / b_p \\ \beta_n & \langle \varphi_n = \text{Sup}[\Delta a_{pn} + a_{pn}^0 - c_{n-1}] / b_p \end{cases} \text{ and } \varphi_{n+1} = \begin{cases} \alpha_{n+1} & \langle \varphi_{n+1} = \text{Inf}[-N(k)] / b_p \\ \beta_{n+1} & \langle \varphi_{n+1} = \text{Sup}[-N(k)] / b_p \end{cases}. \end{aligned} \quad (15)$$

For satisfying (7), if the sampling interval  $T$  is enough small, one can be obtain :

$$L_1 \langle \varphi_1 \rangle M_1, \quad L_i \langle \varphi_i \rangle M_i, \quad i=2, \dots, n-1, \quad L_n \langle \varphi_n \rangle M_n \text{ and } L_{n+1} \langle \varphi_{n+1} \rangle M_{n+1} \quad (16)$$

where:

$$\begin{aligned} M_1 = & \inf \{ [\Delta a_{p1} - a_{p1}^0 \Delta b_p / b_p^0 + c_{i-1} \Delta b_p / b_p^0 - c_1(c_{n-1} - a_{pn}^0)(1 + \Delta b_p / b_p^0) + \gamma_1 / (T |e_1(k) - K_1\dot{z}(k) - K_2z(k) - U_m(k)|)] / b_p \} \\ L_1 = & \sup \{ [\Delta a_{p1} - a_{p1}^0 \Delta b_p / b_p^0 + c_{i-1} \Delta b_p / b_p^0 - c_1(c_{n-1} - a_{pn}^0)(1 + \Delta b_p / b_p^0) - \gamma_1 / (T |e_1(k) - K_1\dot{z}(k) - K_2z(k) - U_m(k)|)] / b_p \} \\ M_i = & \inf \{ [\Delta a_{pi} - a_{pi}^0 \Delta b_p / b_p^0 + c_{i-1} \Delta b_p / b_p^0 - c_i(c_{n-1} - a_{pn}^0)(1 + \Delta b_p / b_p^0) + \gamma_i / (T |x_i(k)|)] / b_p \} \\ L_i = & \sup \{ [\Delta a_{pi} - a_{pi}^0 \Delta b_p / b_p^0 + c_{i-1} \Delta b_p / b_p^0 - c_i(c_{n-1} - a_{pn}^0)(1 + \Delta b_p / b_p^0) - \gamma_i / (T |x_i(k)|)] / b_p \} \\ M_n = & \inf \{ [\Delta a_{pn} - a_{pn}^0 - c_{n-1} + \gamma_n / (T |e_n(k)|)] / b_p, L_n = \sup \{ [\Delta a_{pn} - a_{pn}^0 - c_{n-1} - \gamma_n / (T |e_n(k)|)] / b_p \} \\ M_{n+1} = & \inf \{ [N(k) + \gamma_{n+1} / T] / b_p, L_{n+1} = \sup \{ [N(k) - \gamma_{n+1} / T] / b_p \} \end{aligned}$$

in which  $\gamma_i, i=1, \dots, n+1$ , are positive constants and  $\gamma_1 + \dots + \gamma_{n+1} < \xi / 2$ .

From (15) and (16), if the  $T$  is enough small, one can obtain the bound of  $\alpha_i$  and  $\beta_i$  as :

$$L_i < \alpha_i < \varphi L_i, \quad \varphi U_i < \beta_i < M_i, \quad i = 1, \dots, n+1. \quad (17)$$

If  $\varphi_i, i=1, \dots, n+1$ , are chosen as  $\varphi_i < \alpha_i < -\beta_i$ , finally, the control can be represented as

$$U_p = \left\{ -c_1(K_1 \dot{z}(k) + K_2 z(k)) - \sum_{i=2}^{n-1} c_i e_i(k) + \sum_{i=1}^{n-1} d_{pi}^0 e_i(k) - \sum_{i=1}^n (a_{mi} - d_{pi}^0) x_{mi}(k) + b_m U_m(k) + (c_{n-1} - d_{pm}^0) [c_1(e_1(k) - K_1 \dot{z}(k) - K_2 z(k) - U_m(k)) + \sum_{i=2}^{n-1} c_i e_i(k)] \right\} / b_p^0 + (\varphi [e_1(k) - K_1 \dot{z}(k) - K_2 z(k) - U_m(k)] + \sum_{i=2}^n \varphi_i [e_i(k) + \varphi_{n+1} \text{sig}(\sigma(k))]) \quad (18)$$

From (17), the upper and low bounds of the  $\varphi_i$ , can be obtain

$$\hat{L}_i < \hat{\varphi}_i < \hat{M}_i, \quad i=1, \dots, n+1, \text{ where } \hat{M}_i = -\max(|\varphi L_i|, |\varphi U_i|) \text{ and } \hat{L}_i = -\max(|L_i|, |M_i|) \quad (19)$$

### Determination of switching plane and integral control gain

In the above subsection it has been proved that if the solution of the ideal sliding motion is asymptotically stable, it will close to the solution of the non-ideal sliding within the range of  $\xi$ . Thus one can choose the switching plane and integral control gain of the basis of the ideal sliding motion. While in the ideal sliding motion, the system can described by (4) can be reduced to :

$$\dot{z}(k+1) = \dot{z}(k) - T e_1(k), \quad e_i(k+1) = e_i(k) + T e_{i+1}(k), \quad i=1, \dots, n-2 \quad (20a)$$

$$\dot{e}_n(k+1) = e_{n-1}(k) - T [C_1 K_1 \dot{z}(k) - C_2 K_2 z(k)] - \sum_{i=1}^n a_{pi} e_i(k) \quad (20b)$$

The characteristic equation when the system is on the sliding surface can be shown as

$$H(s) = \frac{E_1(z)}{U_m(z)} = \frac{\left(\frac{z-1}{T}\right)^2 c_1}{\left(\frac{z-1}{T}\right)^n + c_{n-1} \left(\frac{z-1}{T}\right)^{n-1} + \dots + c_1 K_1 \left(\frac{z-1}{T}\right) + c_1 K_2}. \quad (21)$$

The characteristic equation of the system (21) is

$$\left(\frac{z-1}{T}\right)^n + c_{n-1} \left(\frac{z-1}{T}\right)^{n-1} + \dots + c_1 K_1 \left(\frac{z-1}{T}\right) + c_1 K_2 = 0. \quad (22)$$

Since this characteristic equation is independent of the plant parameter, the **DSMFC** approach is robust to plant parameter variations. Further, one can choose the coefficients of the switching function and the integral control gain by the pole assignment technique such that this sliding motion has desirable properties.

Let  $z-1/T = \eta$ . Then (22) can be rewritten as

$$\eta^n + C_{n-1}\eta^{n-1} + \dots + C_1K_1\eta + C_2K_2 = 0. \quad (23)$$

Using the final value theorem, it can be shown from (21), that the steady-state tracking error due to a ramp command input is zero. The transient response of the system can be determined by suitably selecting the poles of the transfer function (21).

$$\text{Let} \quad \eta^n + \alpha_1\eta^{n-1} + \dots + \alpha_{n-1}\eta + \alpha_n = 0 \quad (24)$$

be the desired characteristic equation(closed-loop poles), the coefficient  $C_1, C_2$  and  $K_1, K_2$  can be obtained by

$$C_{n-1} = \alpha_1, C_1 = \alpha_{n-2}, K_1 = \alpha_{n-1}/C_1 \text{ and } K_2 = \alpha_n/C_1.$$

### Chattering Considerations

Normally, the sign function  $\text{sign}(\sigma)$  given by (18), will give rise to chattering in the control signal. To reduce the chattering, the sign function can be replaced by the continuous function, given by

$$M_\delta(\sigma) = \frac{\sigma(k)}{|\sigma(k)| + \delta_0 + \delta_1|\dot{\sigma}(k)|} \text{ where } \delta = \delta_0 + \delta_1|\dot{\sigma}(k)|; \delta_0 \text{ and } \delta_1 \text{ are positive constants.} \quad (25)$$

## Dynamic Modeling of an Induction Motor

The mathematical modeling of a 3-phase, Y-connected squirrel-cage induction motor in the d-q synchronous rotating reference frame can be described by the following state equation :

$$\begin{bmatrix} \dot{v}_{qs}^e \\ \dot{v}_{ds}^e \\ 0 \\ 0 \end{bmatrix} = \begin{bmatrix} r_s + pL_{ss} & \omega_e L_{ss} & pM & \omega_e M \\ -\omega_e L_{ss} & r_s + pL_{ss} & -\omega_e M & pM \\ pM & (\omega_e - \omega_r)M & r_r' + pL_{rr}' & (\omega_e - \omega_r)L_{rr}' \\ -(\omega_e - \omega_r) & pM & -(\omega_e - \omega_r)L_{rr}' & r_r' + pL_{rr}' \end{bmatrix} \begin{bmatrix} i_{qs}^e \\ i_{ds}^e \\ i_{qr}^e \\ i_{dr}^e \end{bmatrix} \quad (26a)$$

$$T_e = \frac{3}{4} pM (i_{qs}^e i_{dr}^e - i_{ds}^e i_{qr}^e) \quad (26b)$$

where

$r_s, r_r'$  is the stator resistance and rotor resistance per phase referred to stator;  $L_{ss}, L_{rr}'$  is the stator inductance and rotor inductance per phase referred to stator;  $P$  is the number of poles;  $M$  is the mutual magnetizing induction;  $P$  is the differential operator;  $v_{qs}^e, (v_{ds}^e)$  is the  $q$ -axis ( $d$ -axis) stator voltage;  $i_{qs}^e, (i_{ds}^e)$  is the  $q$ -axis ( $d$ -axis) stator current;  $i_{qr}^e, (i_{dr}^e)$  is the referred  $q$ -axis ( $d$ -axis) stator current;  $\omega_r$  is the rotor electrical angular velocity and  $\omega_e$  is the synchronous angular velocity. The generated torque, rotor mechanical angular velocity  $\omega_m$  and load can be express as

$$T_e = J_m p \omega_m + B_m \omega_m + T_L \quad (27)$$





and where  $x_{p1} = \theta_m$  is mechanical angular angle of the rotor;  $x_{p2}$  is mechanical angular velocity of the rotor;  $x_{p3}$  is mechanical angular acceleration of the rotor;  $U_p$  is control input and  $U_m = \theta_c$  is desired position.

The reference model is chosen as

$$\dot{x}_{m1} = x_{m2}, \quad \dot{x}_{m2} = x_{m3} \quad \text{and} \quad \dot{x}_{m3} = -a_{m1}x_{m1} - a_{m2}x_{m2} - a_{m3}x_{m3} + b_m U_m. \quad (30)$$

Defining  $e_i = x_{pi} - x_{mi}$  ;( $i= 1,2,3$ ), the DSMFC system can be represented as

$$\dot{z}(k+1) = \dot{z}(k) - Te_1(k), \quad e_2(k+1) = e_2(k) + Te_3(k) \quad ; i=1, \dots, n-1 \quad (31a)$$

$$e_3(k+1) = e_3(k) - T \sum_{i=1}^n a_{pi} e_i(k) - T \sum_{i=1}^n (a_{mi} - a_{pi}) x_{mi}(k) + Tb_m U_m(k) - Tb_p U_p(k) - Tf(k) \quad (31b)$$

Following the design procedure we have the control law to implement as

$$\begin{aligned} U_p(k) = & \left\{ c_1(K_1 \ddot{z}(k) + K_2 \dot{z}(k)) + a_{p1}^0 e_1(k) + a_{p2}^0 e_2(k) - [(a_{m1} - a_{p1}^0)x_{m1}(k) + (a_{m2} - a_{p2}^0)x_{m2}(k) + (a_{m3} - a_{p3}^0)x_{m3}(k) + b_m U_m] \right. \\ & + (c_2 - a_{p1}^0) [c_1(e_1(k) - K_1 \dot{z}(k) - K_2 z(k) - r(k)) + c_2 e_2(k)] \Big\} / b^0 + (\Psi_1' e_1(k) - K_1 \dot{z}(k) - K_2 z(k) - r(k)) + \Psi_2' e_2(k) \\ & + \Psi_3' e_3(k) + \Psi_4) M_\sigma(\sigma) \end{aligned} \quad (32)$$

The switching function,  $\sigma(k)$  from (6), is given by

$$\sigma(k) = c_1(e_1(k) - K_1 \dot{z}(k) - K_2 z(k) - r(k)) + c_2 e_2(k) + e_3(k). \quad (33)$$



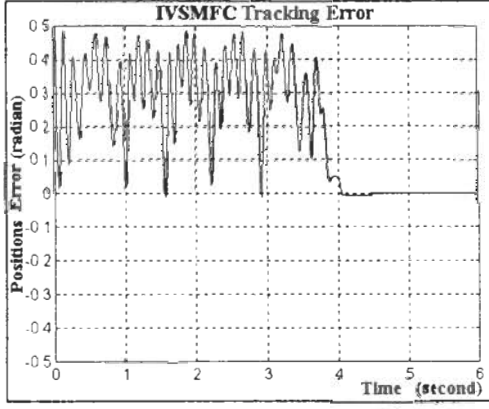
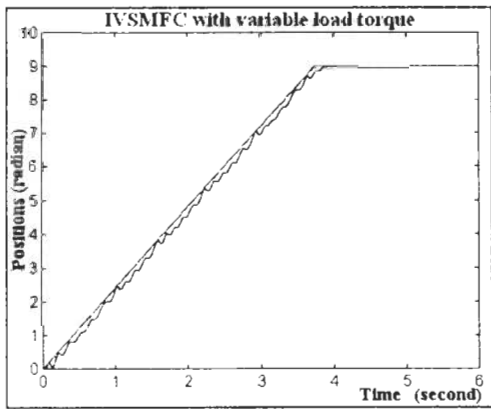
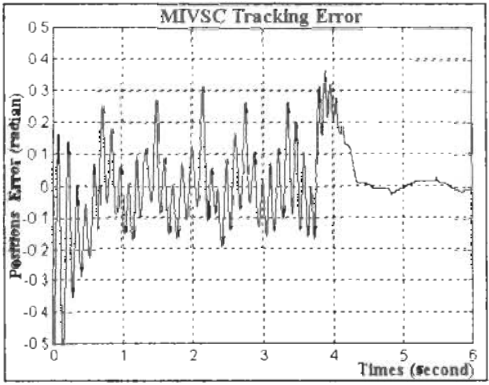
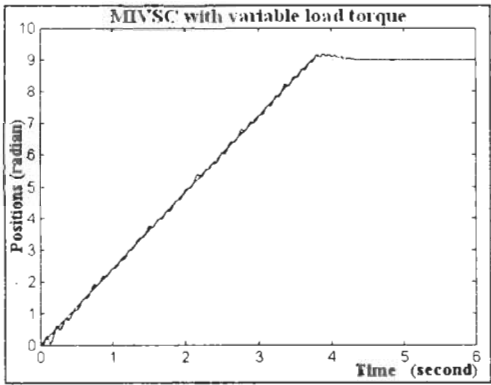
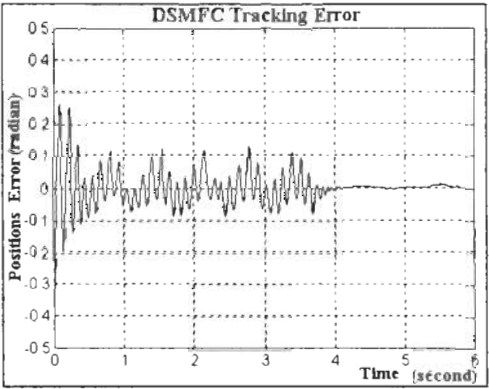
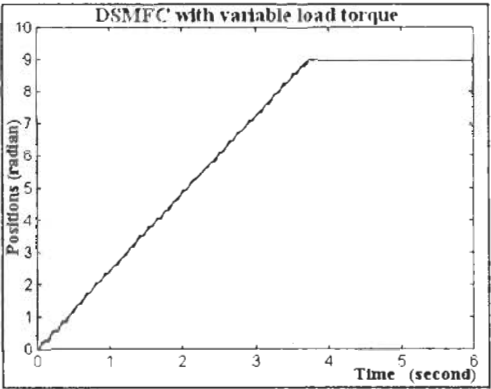


Figure 5 Comparison of ramp position tracking

Figure 6 Comparison of position tracking errors

## Conclusions

An induction motor position control driver using DSMFC approach is presented. It has been developed a procedure for determining the control function and switching plane by using model following control. Also, by using a continuous function, the chattering can be effectively suppressed. The application of SMFC to the induction motor position control system has illustrated that the DSMFC approach can improve the tracking performance by 70% and 80% when compared to the MIVSC and IVSMFC approaches, respectively. Experimental results demonstrate that the proposed approach can achieve accurate position tracking in face of wide plant parameter variations and external load disturbance. Furthermore, The DSMFC approach is robust and applicable to electrical machine control systems. It is therefore expected that the proposed control scheme can be applied to the high performance applications of induction motor such as the high precision control of machine tools and industrials.

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